University of Nebraska - Lincoln

DigitalCommons@University of Nebraska - Lincoln

Nebraska Tractor Tests

Tractor Test and Power Museum, The Lester F. Larsen

1-1-1973

Test 1150: Case 870 Power Shift Diesel

Nebraska Tractor Test Lab University of Nebraska-Lincoln, tractortestlab@unl.edu

Follow this and additional works at: https://digitalcommons.unl.edu/tractormuseumlit

Part of the Energy Systems Commons, History of Science, Technology, and Medicine Commons, Other Mechanical Engineering Commons, Physical Sciences and Mathematics Commons, Science and Mathematics Education Commons, and the United States History Commons

Nebraska Tractor Test Lab, "Test 1150: Case 870 Power Shift Diesel" (1973). *Nebraska Tractor Tests*. 1473.

https://digitalcommons.unl.edu/tractormuseumlit/1473

This Article is brought to you for free and open access by the Tractor Test and Power Museum, The Lester F. Larsen at DigitalCommons@University of Nebraska - Lincoln. It has been accepted for inclusion in Nebraska Tractor Tests by an authorized administrator of DigitalCommons@University of Nebraska - Lincoln.

NEBRASKA TRACTOR TEST 1150 - CASE 870 POWER SHIFT DIESEL

			PC	OWER	TAKI	E-OFF	PER	FORM	IANC	E			
			Crank- shaft	Fuel Consumpti Gal Lb			Ip-hr	Tempe	erature D Air	ure Degrees F Air Air		Barometer	
	Hp		speed	per	per		per	Cooling	wet	dr	У	inches of Mercury	
_			rpm	hr IMUM	hp-h		gal				.b	Wercury	
											`		
	80.4		2100	Engine S 6.040	0.51	9 1	3.33	208	68	75		28.920	
	76.52		Standa 1870	rd Powe 5.610	er Take 0.50		ed (10 3.64	000 rpm 209) –One 70	Hour 75	5	28.890	
		VA	RYING	POWE	RAND.	FUEL	CONS	SUMPT	ION-T	wo H	ours	1	
	71.5		2191	5.120	0.49		3.93	193	70	75			
_	0.0	00	2286	1.652				178	70	74	1		
-	36.35		2230	3.113	0.59	3 1	1.68	184	73	78	3		
	80.11		2100 6.100		0.52	27 1.	3.13	205	72	75	5		
	18.3	34	2252	2.397	0.90)5	7.65	179	70	74	ŧ		
li .	54.0)5	2213	4.049	0.51		3.35	186	70	7-			
Av	43.3	37	2212	3.738	0.59	6 1	1.60	187	71	7:)	23.857	
				DRA	WBA]	R PEF	RFOF	RMAN	CE				
I.	Hp p		mile per	speed	Slip of drivers	Fuel Cons Gal per hr	sumption Lb per hp-hr	Hp-hr per	Temp Cool- ing med	Degre Air wet bulb	es F Air dry bulb	Baromete inches of Mercury	
	DATE	lbs	hr	rpm	% MED 4:			gal					
VA.	KYI.	NG I										BALLAST	
68.	.90	5840	4.42		7.25	5.997	0.602	11.49	189	45	51	28.540	
57.	.65	4558		ull at M 2211	aximum 5.51	4.790	—Ten 0.575		6th Ge: 184	ar (3 I 44	. o) 51	28.570	
39	.82	50 3052		ull at M 2235	aximum 3.67	Power- 3.757	-Two 0.653		- 6th Ge 182	ar (3 I	L o) 49	28.530	
39	.09	0% o 2996		at Redu 1673	ced Eng	ine Spe 3.106	ed—T 0.549		rs—8th 181	Gear 45	(3) 52	Int) 28.480	
				MAXII	MUM P	OWER	WIT	H BAL	LAST				
61	.15	10030	2.29	2198	14.98	2nd G	ear (1	Int)	184	41	44	28.590	
70	.32	6864	3.84	2101	9.29	5th G	ear (2	Int)	192	48	55	28.640	
71	.04	6072	4.39	2098	7.98	6th G	ear (3	Lo)	191	48	55	28.640	
68	.36	5202	4.93	2099	6.63	7th G	ear (2	Hi)	192	48	55	28.640	
70	.26	_4394			5.60	8th G	,		192	47	53	28.650	
66	.28	2432			3.08	10th (192	47	53	28.650	
	VA	RYIN	NG DR	AWBAR		AND T h Gear			ED WI	THE	AL	LAST	
Pou	ınds	Pull	l		6072	6482	6	6809	6983	699	18	6894	
Horsepow		ower	er		71.04	67.87	62	2.90	56.54	48.13		39.23	
Crankshaft		aft S	Speed r	pm	2098	1891	1	.681	1479	123	57	1039	
Mil	es P	er H	lour		4.39	3.93	3.46		3.04	2.58		2.13	
Slip of Drivers %			7.98	8.71		9.23	9.61	9.61		9.61			
					RACTO		JND 1	LEVEL			dB	(A)	
Maximum Available Power 2 Hours										82.0			
75% of Pull at Max. Power 10 Hours										84.0			
50% of Pull at Max. Power 2 Hours										83.5			
	50% of Pull at Reduced Engine Speed 2 Hours								3 1	82.0			
Bystander 12th Gear (4-Hi)										86.5			
				AND W				Ballast				Ballast	
F		Tire llast	es	-No., size, ply & psi Two 18.4-34;8;20 -Liquid 998 lb each						Two 18.4-34;8;20 None			

Front Tires

Ballast

Height of drawbar

Static weight with operator-rear

Cast Iron

Cast Iron

front

total

—Liquid

Department of Agricultural Engineering Dates of Test: October 26th to November 6,

Manufacturer:

anufacturer: J. I. CASE COMPANY, RACINE, WISCONSIN
FUEL, OIL AND TIME Fuel No 2 Diesel Cetane No 50.1 (rating taken from oil company's typical inspection data) Specific gravity converted to 60°/60° 0.8311 Weight per gallon 6.920 lb Oil SAE 30 API service classification (Case HDM Oil) To motor 1.920 gal Drained from motor 1.626 gal Transmission and final drive lubricant Case TFD Oil Total time

engine was operated 48 hours.

ENGINE Make J. I. Case Diesel

Cylinder vertical Serial No 2531468

Crankshaft Mounted lengthwise Rated rpm 2100 Bore and stroke 45/8'' x 5" Compression ratio 16.5 to 1 Displacement 336 cu in Cranking system 12 volt electric Lubrication pressure Air cleaner dry type with replaceable pleated paper element and precleaner Oil filter Full flow replaceable cartridge Oil cooler engine coolant heat exchanger for engine oil and radiator for transmission and hydraulic oil Fuel filter replaceable primary and secondary filter cartridges Muffler was used Cooling medium

temperature control thermostat.

CHASSIS Type standard Serial No 8728083 Tread width rear 62" to 88" front 62" to 90" Wheel base 101" Center of gravity (without operator or ballast, with minimum tread, with fuel tank filled and tractor serviced for operation) Horizontal distance forward from center-line of rear wheels 29.5" Vertical distance above roadway 39.1" Horizontal distance from center of rear wheel tread 0" to the right/left Hydraulic control system direct engine drive Transmission selective gear fixed ratio with partial range operator controlled power shifting Advertised speeds mph first 1.8 second 2.5 third 3.0 fourth 3.1 fifth 4.0 sixth 4.6 seventh 5.0 eighth 6.2 ninth 7.7 tenth 10.2 eleventh 13.7 twelfth 17.0 reverse 3.1, 5.0, 7.7 Clutch multiple disc wet clutches within transmission actuated hydraulically Brakes dry double disc hydraulically power actuated by two foot pedals that can be locked together Steering hydrostatic Turning radius (on concrete surface with brake applied) right 156" left 156" (on concrete surface without brake) right 182" left 182" **Turning** space diameter (on concrete surface with brake applied) right 320" left 320" (on concrete surface without brake) right 373" left 373" Belt pulley 1107 rpm at 1900 engine rpm diam 10.5" face 7.25" Belt speed fpm 3045 Power take-off 538 rpm at 1900 engine rpm and 1016 rpm at 1900 engine rpm.

REPAIRS AND ADJUSTMENTS No repairs

or adjustments.

REMARKS: All test results were determined from observed data optained in accordance with SAE and ASAE test code or official Nebraska test procedure.

First gear was not run as it was necessary to limit the pull in second gear to avoid excessive wheel slippage. Third, fourth, ninth, eleventh and twelfth gears were not run as test procedure permits a maximum of six travel speeds.

We, the undersigned, certify that this is a true and correct report of official Tractor Test 1150. L. F. LARSEN

Engineer-in-Charge
G. W. STEINBRUEGGE, Chairman W. E. SPLINTER D. E. LANE Board of Tractor Test Engineers

2900 lb

10095 lb

None

None

None

Two 10.00-16;6;28

 $21\frac{1}{2}$ inches 7195 lb

720 lb each

115 lb each

201/2 inches

10630 lb

3130 lb

13760 lb

-No., size, ply & psi Two 10.00-16;6;28

None

EXPLANATION OF TEST REPORT

GENERAL CONDITIONS

Each tractor is a production model equipped for common usage. Power consuming accessories can be disconnected only when it is convenient for the operator to do so in practice. Additional weight can be added as ballast if the manufacturer regularly supplies it for sale. The static tire loads and the inflation pressures must conform to recommendations in the Tire Standards published by the Society of Automotive Engineers.

PREPARATION FOR PERFORMANCE RUNS

The engine crankcase is drained and refilled with a measured amount of new oil conforming to specifications in the operators manual. The fuel used and the maintenance operations must also conform to the published information delivered with the tractor. The tractor is then limbered-up for 12 hours on drawbar work in accordance with the manufacturer's published recommendations. The manufacturer's representative is present to make appropriate decisions regarding mechanical adjustments.

The tractor is equipped with approximately the amount of added ballast that is used during maximum drawbar tests. Prior to the maximum power run the tire tread-bar height must be at least 65% of new tread height.

POWER TAKE-OFF PERFORMANCE

Maximum Power and Fuel Consumption. The manufacturer's representative makes carburetor, fuel pump, ignition and governor control settings which remain unchanged throughout all subsequent runs. The governor and the manually operated governor control lever is set to provide the high-idle speed specified by the manufacturer for maximum power. Maximum power is measured by connecting the power take-off to a dynamometer. The dynamometer load is then gradually increased until the engine is operating at the rated speed specified by the manufacturer for maximum power. The corresponding fuel consumption is measured.

Varying Power and Fuel Consumption. Six different horsepower levels are used to show corresponding fuel consumption rates and how the governor causes the engine to react to the following changes in dynamometer load: 85% of the dynamometer torque at maximum power; minimum dynamometer torque, ½ of the 85% torque; maximum power, ¼ and ¾ of the 85% torque. Since a tractor is generally subjected to varying loads the average of the results in this test serve well for predicting the fuel consumption of a tractor in general usage.

DRAWBAR PERFORMANCE

All engine adjustments are the same as those used in the belt or power take-off tests.

Varying Power and Fuel Consumption With Ballast. The varying power runs are made to show the effects of speed-control devices (engine, governor, automatic transmission, etc.) on horsepower, speed and fuel consumption. These runs are made around the entire test course which has two 180 degree turns with a minimum radius of 50 feet. The drawbar pull is set at 4 different runs as follows: (1) as near to the pull at maximum power as

possible and still have the tractor maintain the travel speed at maximum horsepower on the straight sections of the test course; (2) 75% of the pull at maximum power; (3) 50% of the pull at maximum power; and (4) maintaining the same load and travel speed as in (3) by shifting to a higher gear and reducing the engine rpm.

Maximum Power with Ballast. Maximum power is measured on straight level sections of the test course. Data are shown for not more than 6 different gears or travel speeds. Some gears or travel speeds may be omitted because of high slippage of the traction members or because the travel speed may exceed the safe limit for the test course. The manufacturer's representative has the option of selecting one gear or speed over eight miles per hour. The maximum safe speed for the Nebraska Test Course has been set at 15 miles per hour. The slippage limits have been set at 15% and 7% for pneumatic tires and steel tracks or lugs, respectively. Higher slippage gives widely varying results.

Varying Drawbar Pull and Travel Speed with Ballast. Travel speeds corresponding to drawbar pulls beyond the maximum power range are obtained to show the "lugging ability" of the tractor. The run starts with the pull at maximum power; then additional drawbar pull is applied to cause decreasing speeds. The run is ended by one of three conditions: (1) maximum pull is obtained, (2) the maximum slippage limit is reached, or (3) some other operating limit is reached.

SOUND MEASUREMENT

Sound is recorded during each of the Varying Power and Fuel Consumption runs as the tractor travels on a straight section of the test course. The dB(A) sound level is obtained with the microphone located near the right ear of the operator. Bystander sound readings are taken with the microphone placed 25 feet from the line of travel of the tractor.

An increase of 10 dB(A) will approximately double the loudness to the human ear.

For additional information about the Nebraska Tractor Tests write to the Department of Agricultural Engineering, University of Nebraska, Lincoln, Nebraska 68503.



CASE 870 POWER SHIFT DIESEL